

Article

Effect of Wall Boundary Conditions on a Wall-Modeled Large-Eddy Simulation in a Finite-Difference Framework

H. Jane Bae ^{1,2,*}  and Adrián Lozano-Durán ³ ¹ School of Engineering and Applied Sciences, Harvard University, Cambridge, MA 02138, USA² Graduate Aerospace Laboratories, California Institute of Technology, Pasadena, CA 91125, USA³ Department of Aeronautics and Astronautics, Massachusetts Institute of Technology, Cambridge, MA 02139, USA; adrianld@mit.edu

* Correspondence: jbae@caltech.edu

Abstract: We studied the effect of wall boundary conditions on the statistics in a wall-modeled large-eddy simulation (WMLES) of turbulent channel flows. Three different forms of the boundary condition based on the mean stress-balance equations were used to supply the correct mean wall shear stress for a wide range of Reynolds numbers and grid resolutions applicable to WMLES. In addition to the widely used Neumann boundary condition at the wall, we considered a case with a no-slip condition at the wall in which the wall stress was imposed by adjusting the value of the eddy viscosity at the wall. The results showed that the type of boundary condition utilized had an impact on the statistics (e.g., mean velocity profile and turbulence intensities) in the vicinity of the wall, especially at the first off-wall grid point. Augmenting the eddy viscosity at the wall resulted in improved predictions of statistics in the near-wall region, which should allow the use of information from the first off-wall grid point for wall models without additional spatial or temporal filtering. This boundary condition is easy to implement and provides a simple solution to the well-known log-layer mismatch in WMLES.

Keywords: wall modeling; turbulence; boundary layer



Citation: Bae, H. J.; Lozano-Durán, A. Effect of Wall Boundary Conditions on a Wall-Modeled Large-Eddy Simulation in a Finite-Difference Framework. *Fluids* **2021**, *6*, 112. <https://doi.org/10.3390/fluids6030112>

Academic Editors: Mehrdad Massoudi and Timothy Wei

Received: 11 February 2021

Accepted: 5 March 2021

Published: 10 March 2021

Publisher's Note: MDPI stays neutral with regard to jurisdictional claims in published maps and institutional affiliations.



Copyright: © 2021 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (<https://creativecommons.org/licenses/by/4.0/>).

1. Introduction

The near-wall grid resolution required to accurately resolve the boundary layer in wall-bounded flows remains a pacing item in large-eddy simulations (LES) for high-Reynolds-number engineering applications. Studies by Chapman [1], and more recently by Choi and Moin [2] and Yang and Griffin [3], estimated that the number of grid points necessary for wall-resolved LES scales is $Re^{13/7}$, where Re is the characteristic Reynolds number of the problem. The computational cost is still excessive for many practical problems, especially for external aerodynamics, despite the favorable comparison with the $Re^{37/14}$ scaling required for direct numerical simulation (DNS), where all the relevant scales of motion are resolved. Even in LES, the large-scale structures relevant to wall-bounded turbulence, as observed since the early days of flow visualization [4–7], need to be adequately resolved in order to obtain accurate predictions using simulations. The alternative to resolving the large-scale structures of the near-wall region is wall modeling. By modeling the near-wall flow such that only the large-scale motions in the outer region of the boundary layer are resolved, the grid-point requirement for wall-modeled LES scales at most linearly with increasing Reynolds number. Therefore, wall-modeling stands as the most feasible approach for wall-resolved LES or DNS.

Examples of the most popular and well-known wall models are the traditional wall-stress models (or approximate boundary conditions) and the detached eddy simulation (DES) and its variants. DES [8] combines RANS equations close to the wall and LES in the outer layer with the interface between Reynolds-averaged Navier–Stokes and LES domains enforced implicitly through the change in the turbulence model. The traditional

wall-stress models impose an approximate boundary condition at the wall rather than a no-slip boundary condition, imposing a wall stress (computed by the model) or the law of the wall through the boundary condition. Deardorff [9] used a boundary condition based on the law of the wall, imposing a condition on the second derivative of the streamwise velocity in the wall-normal direction. Schumann [10] used a finite volume approach and a staggered grid and supplied the correct tangential shear stress at the wall as the boundary flux term. Throughout the years, many researchers followed the method of Schumann [10] for finite difference methods by utilizing a Neumann boundary condition in the wall-parallel velocity components to specify the shear stress at the wall due to the straightforward connection between the wall shear stress and the wall-normal component of the streamwise velocity gradient at the wall [11–14]. More recently, Robin boundary conditions in the three velocity components have been employed in wall-modeled large-eddy simulations (WMLES) [15,16]. The reader is referred to Cabot and Moin [12], Piomelli and Balaras [17], Larsson et al. [18] and Bose and Park [19] for a more comprehensive review of wall-stress models.

In most of the instances mentioned above, the eddy viscosity at the wall is assumed to be zero or computed from the subgrid-scale (SGS) model, and is not part of the wall model. Using the eddy viscosity at the wall as a wall-modeling term provides various options in imposing the correct wall shear stress through the stress-balance equation. One such option is to impose a boundary condition on the eddy viscosity at the wall rather than the wall-normal derivative of the streamwise velocity [20]. In the current study, we compare three different boundary conditions that arise from the mean LES stress-balance equation at the wall wherein a no-penetration (zero wall-normal velocity at the wall) condition is assumed in a finite-difference framework. We will show that augmenting the eddy-viscosity at the wall, rather than imposing a Neumann boundary condition for the wall-parallel velocities, provides a better prediction of the mean velocity profile and turbulence statistics in the first few grid points off the wall.

Better prediction of the mean velocity profile in the first few grid points off the wall has real-world implications for wall modeling. Log-layer mismatch refers to a well-known problem in WMLES where the modeled wall-shear stress deviates from the actual value, either by underpredicting [14,21] or overpredicting [22,23] the stress. Previous works attempting to solve the issue of log-layer mismatch include works on modifying the SGS stress models [24], using LES information farther away from the wall [14], and filtering the LES information in space or time [25]. We will show that applying the boundary condition to the eddy-viscosity provides a simple method to correct for the log-layer mismatch without the need for spatial or temporal filtering or LES information farther away from the wall, which may be more difficult to obtain in complex geometries.

The paper is organized as follows. In Section 2, we present the three different boundary conditions under consideration and describe the numerical experiments in our analysis. In Section 3, we study the effects of the boundary conditions on the mean velocity profile, the turbulence intensities, and the Reynolds stress tensor for various Reynolds numbers and grid resolutions. Section 4 demonstrates that log-layer mismatch can be resolved using eddy-viscosity augmentation. Finally, in Section 5, we discuss the efficacy of the three boundary conditions and offer conclusions.

2. Numerical Simulations

In the following, we consider a turbulent flow between two parallel walls. The streamwise, wall-normal, and spanwise directions are denoted by x , y , and z (or x_1 , x_2 , and x_3), respectively. The flow velocities in the corresponding directions are given by u , v , and w (or u_1 , u_2 , and u_3), respectively. The flow is characterized by the friction Reynolds number $Re_\tau = u_\tau \delta / \nu$ where δ is the half channel height, u_τ is the friction velocity, and ν is the kinematic viscosity.

We solve the filtered Navier–Stokes equations

$$\frac{\partial \bar{u}_i}{\partial t} + \frac{\partial \bar{u}_i \bar{u}_j}{\partial x_j} = -\frac{1}{\rho} \frac{\partial \bar{p}}{\partial x_i} + \nu \frac{\partial^2 \bar{u}_i}{\partial x_j \partial x_j} + \frac{\partial \tau_{ij}^{SGS}}{\partial x_j}, \quad \frac{\partial \bar{u}_i}{\partial x_i} = 0, \tag{1}$$

where $(\bar{\cdot})$ denotes filtered quantities, ρ is the density, p is the pressure, and τ_{ij}^{SGS} is the deviatoric part of the SGS stress tensor. These equations are solved by means of a staggered second-order finite difference [26] and a fractional-step method [27] with a third-order Runge–Kutta time-advancing scheme [28]. Periodic boundary conditions are imposed in the streamwise and spanwise directions. The SGS stress tensor is approximated using the anisotropic minimum dissipation (AMD) model [29], and in particular, it can be written as $\tau_{ij}^{SGS} = -2\nu_t \bar{s}_{ij}$, where ν_t is the eddy viscosity given by the model and

$$\bar{s}_{ij} = \frac{1}{2} \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right).$$

The code has been validated in previous studies in turbulent channel flows [16,30–32].

In WMLES, the size of the dynamically important energy-containing eddies close to the wall is too small to be resolved by the grid resolution at an affordable cost. Thus, a wall model acts as a surrogate for the missing scales close to the wall by providing the correct wall shear stress τ_w to be applied at the wall boundaries through a boundary condition. In an LES of channel flow with a no-penetration boundary condition for the wall-normal velocity and an eddy viscosity SGS model, the wall stress is given by

$$\frac{\langle \tau_w \rangle}{\rho} = \nu \left\langle \frac{\partial \bar{u}}{\partial y} \Big|_w \right\rangle - \langle \tau_{12}^{SGS} |_w \rangle \tag{2}$$

$$= \left\langle (\nu + \nu_t) \frac{\partial \bar{u}}{\partial y} \Big|_w \right\rangle, \tag{3}$$

where ν_t is the eddy viscosity. The subscript w indicates quantities evaluated at the wall and $\langle \cdot \rangle$ indicates averaging in homogeneous directions and time. Fluctuating quantities with respect to the mean will be indicated with $(\cdot)'$. Note that some authors utilize a non-zero eddy viscosity at the wall by computing its value from the SGS model at hand, whereas others have argued that ν_t should be set to zero at the wall as the wall model is solely responsible for generating the wall stress.

There are two main methods to apply the wall stress locally in space and time. The first method is to utilize a Neumann boundary condition [10,12–14,33] where

$$\frac{\partial \bar{u}}{\partial y} \Big|_w = \frac{\tau_w}{\rho(\nu + \nu_t)|_w}. \tag{4}$$

This boundary condition may be subdivided according to whether the eddy viscosity ν_t is set to zero at the wall or assumed to have a natural extension at the wall. An alternate method is to augment the eddy viscosity at the wall [20] such that

$$\nu_t|_w = \left(\frac{\partial \bar{u}}{\partial y} \right) \Big|_w^{-1} \frac{\tau_w}{\rho} - \nu \tag{5}$$

while applying a Dirichlet boundary condition for the velocity components. Equations (4) and (5) are straightforward to apply in a finite difference code once all the necessary components on the right-hand-side are evaluated at the wall. Additional boundary conditions which assume a nonzero wall-normal velocity at the wall [15,16,30], which adds additional terms to Equation (3), can be imposed; however, these boundary conditions are outside the scope of the current paper. The wall boundary conditions under consideration are

- Neumann boundary condition with zero eddy viscosity at the wall (N-ZEV):

$$\left. \frac{\partial \bar{u}}{\partial y} \right|_w = \frac{\tau_w}{\rho \nu}, \quad \bar{v}|_w = 0, \quad \left. \frac{\partial \bar{w}}{\partial y} \right|_w = 0, \quad \nu_t|_w = 0, \quad (6)$$

- A Neumann boundary condition with nonzero eddy viscosity computed from the SGS model at the wall (N-EV):

$$\left. \frac{\partial \bar{u}}{\partial y} \right|_w = \frac{\tau_w}{\rho(\nu + \nu_t)|_w}, \quad \bar{v}|_w = 0, \quad \left. \frac{\partial \bar{w}}{\partial y} \right|_w = 0, \quad (7)$$

- A Dirichlet boundary condition with eddy viscosity augmentation (D-EV):

$$\bar{u}|_w = 0, \quad \bar{v}|_w = 0, \quad \bar{w}|_w = 0, \quad \nu_t|_w = \left(\left. \frac{\partial \bar{u}}{\partial y} \right|_w \right)^{-1} \frac{\tau_w}{\rho} - \nu. \quad (8)$$

In the cases of D-EV, the eddy viscosity at the cell-center of the first off-wall grid point is set to the desired value at the wall. In both N-EV and D-EV, the value of ν_t at the first off-wall grid point is then extrapolated to the wall.

In the following section, we perform an idealized WMLES of a channel flow at $Re_\tau = 5200, 10^4$ and 10^5 for each of these boundary conditions. To isolate the effect of the boundary condition, we supply the exact wall shear stress, $\tau_w = \rho u_\tau^2$ in the streamwise direction and zero wall stress in the spanwise direction, and the channel flow is driven by imposing a constant mass flow in the resolved logarithmic region, which allows us to simulate Reynolds numbers for which we do not have a DNS mean velocity profile. The numerical domain is $2\pi\delta \times 2\delta \times \pi\delta$ in the streamwise, wall-normal, and spanwise directions, respectively. It has been shown that this domain size is sufficient to accurately predict one-point statistics for Re_τ up to 4200 [34]. The domain is discretized using three different grid resolutions, $128 \times 40 \times 64, 64 \times 20 \times 32,$ and $32 \times 10 \times 16$ grid points in each direction. This corresponds to uniform grid spacings in the three directions of $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.05, 0.1,$ and $0.2,$ which are typical resolutions utilized in WMLES, where $\Delta_{x,y,z}$ corresponds to the grid resolutions in the $x, y,$ and z directions, respectively. The simulations were run for 100 eddy turnover times (defined as δ/u_τ) after transients.

3. Comparison of the Boundary Conditions

As the base case, we compare the results of the three boundary conditions (N-ZEV, N-EV, D-EV) for $Re_\tau = 5200$ at grid resolution $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1.$ The mean velocity profiles are shown in Figure 1. The main difference is observed in the centerline velocity, where D-EV provides a better prediction compared to N-ZEV and N-EV (Figure 1a). The three cases have little difference in the shape of the mean velocity profile in the outer region, as seen from the defect form of the mean velocity profile (Figure 1b). This is consistent with previous observations that the boundary condition does not affect the outer region [16,31,32]. The mean eddy viscosity for the three boundary conditions collapses in the outer region (Figure 2), which is further evidence that the outer region is not affected by the wall boundary condition.

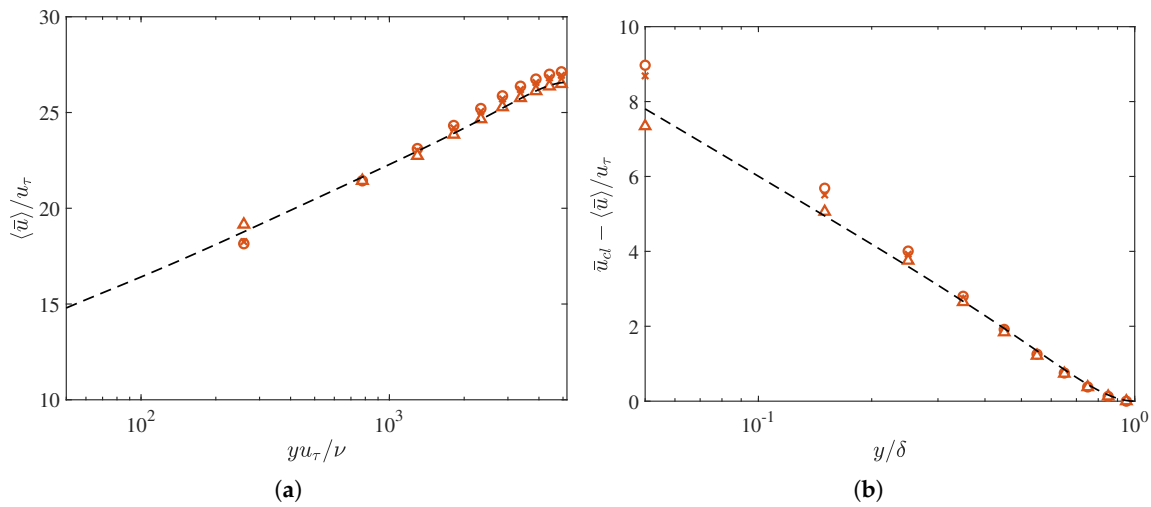


Figure 1. Mean velocity profile in (a) semi-log and (b) defect forms for N-ZEV (circles), N-EV (crosses), and D-EV (triangles) for the $Re_\tau = 5200$ case with $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$. The dashed line indicates the direct numerical simulation (DNS) profile by Lee and Moser [35].

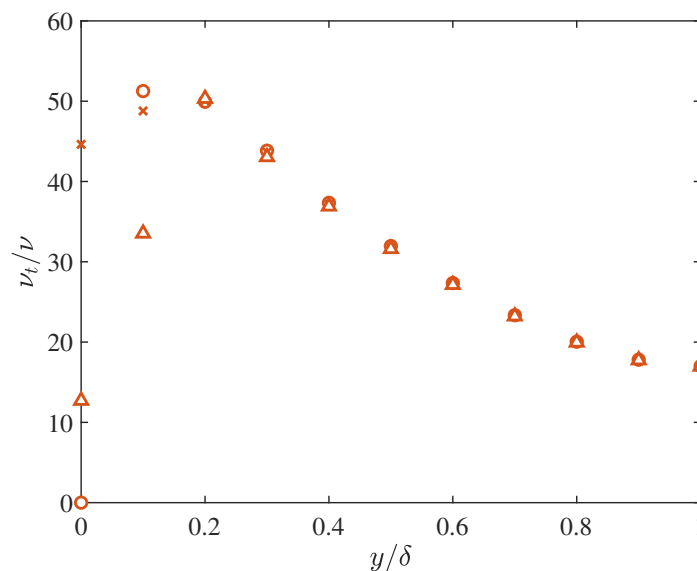


Figure 2. Eddy viscosity as a function of wall-normal distance for N-ZEV (circles), N-EV (crosses), and D-EV (triangles) for the $Re_\tau = 5200$ case with $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$.

The discrepancy in the centerline velocity is due to the way the channel is driven. As the mean mass flow in the logarithmic region is constant, if the mean velocity profile in the logarithmic region (first 1 or 2 grid points off the wall) is underestimated, the mean profile is shifted up as is the case for the boundary conditions N-ZEV and N-EV. The Neumann boundary condition compensates for the missing wall stress by providing a very high streamwise velocity gradient at the wall (Table 1), and the mean profile requires 1 or 2 grid points to counteract the steep gradient. In addition, the eddy viscosity close to the wall is estimated to be larger in the Neumann boundary condition cases (Figure 2 and Table 1), which increases the total SGS shear stress close to the wall, increasing dissipation in this region. While different methods of driving the flow may alleviate the mismatch in the centerline velocity (such as imposing the centerline velocity), this will shift the mean profiles of the N-ZEV and N-EV cases down in a way that will mean the velocity in the first two grid points will be underpredicted. This indicates that D-EV performs better compared to the other two boundary conditions closer to the wall, particularly in the first few grid-points off the wall and at the wall (Table 1).

Table 1. Mean discrete wall normal derivative of the streamwise velocity at the wall, the mean streamwise velocity at the wall, and the eddy viscosity at the first off-wall grid point for the three different boundary conditions at $Re_\tau = 5200$ with grid resolution $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$. DNS results computed with $\Delta_y/\delta = 0.1$.

Case	$\frac{[\langle \bar{u} \rangle(y=\Delta_y/2) - \langle \bar{u} \rangle(y=0)]}{\Delta_y/2} \frac{\delta}{u_\tau}$	$\frac{\langle \bar{u}(y=0) \rangle}{u_\tau}$	$\frac{\langle \nu_t \rangle(y=\Delta_y)}{\nu}$
DNS	375.8	0	-
N-ZEV	5186	-241.1	51.3
N-EV	2130	-88.3	48.8
D-EV	383.0	0	33.2

The resolved turbulence intensities in the form of root-mean-square velocity fluctuations are shown in Figure 3a–c. Similar to the mean velocity profile, the turbulence intensities collapse for the three cases in the outer region, as expected from the collapse of the eddy viscosity in this region (Figure 2). The main difference is observed in the near-wall region, where the D-EV case provides a better estimation of the turbulence intensities in the first two off-wall grid points. There is a marginal difference between the N-ZEV and N-EV cases, with N-EV providing slightly better estimation. Similar trends can be observed for the resolved tangential Reynolds stress, $\bar{u}'v'$ (Figure 3d). This difference can be attributed to the high streamwise velocity gradient at the wall for the Neumann boundary condition cases, which in turn elevates the eddy viscosity close to the wall (as seen in Figure 2) and provides excessive dissipation close to the wall.

The effect of the eddy viscosity can be incorporated to study the turbulence intensities by accounting for the additional stress generated from τ_{ij}^{SGS} . If we take the Reynolds stress tensor for LES to be approximated as

$$R_{ij} = \langle u_i u_j \rangle - \langle u_i \rangle \langle u_j \rangle \approx \langle \bar{u}_i \bar{u}_j \rangle - \langle \bar{u}_i \rangle \langle \bar{u}_j \rangle + \tau_{ij}^{SGS}, \tag{9}$$

we can compare the deviatoric part of R_{ij} , $R_{ij}^D = R_{ij} - 1/3 R_{kk} \delta_{ij}$ for the LES and DNS, where δ_{ij} is the Kronecker delta. The total tangential Reynolds shear stress R_{12} (Figure 4d) shows that by including the effect of the SGS model, the missing stresses can be supplemented. The three diagonal components of R_{ij}^D are shown in Figure 4, which shows that for R_{11}^D and R_{22}^D , D-EV outperforms N-ZEV and N-EV, while results for R_{33}^D are mixed. Nevertheless, in most applications of WMLES, the statistics from the resolved quantities are taken as is without any additional consideration of the contribution from the SGS scales, and the results for R_{ij}^D are meant for qualitative comparison.

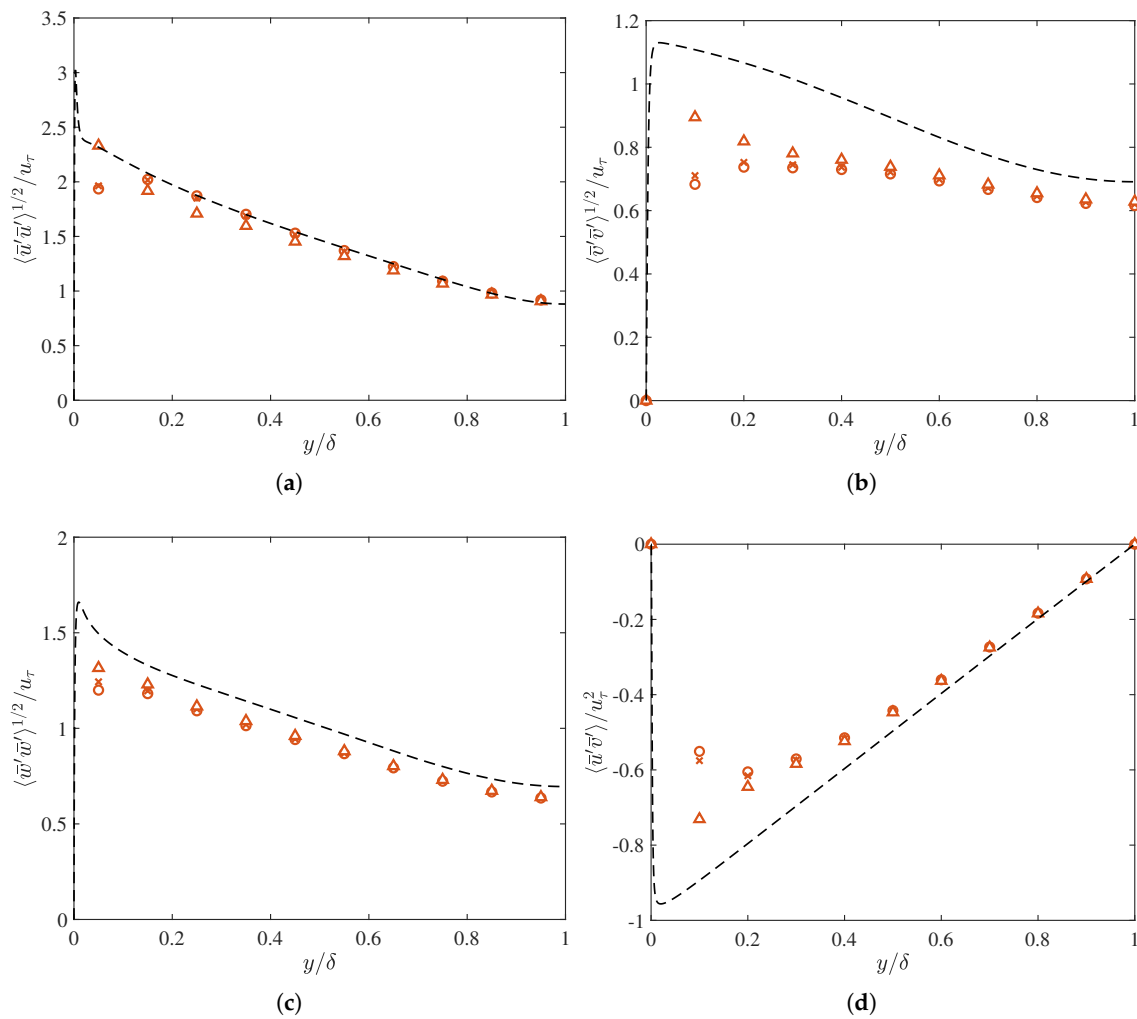


Figure 3. Root-mean-square (a) streamwise, (b) wall-normal, and (c) spanwise velocity fluctuations and the (d) resolved tangential Reynolds stress for N-ZEV (circles), N-EV (crosses), D-EV (triangles) for the $Re_\tau = 5200$ case with $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$. The dashed line indicates the DNS profile by Lee and Moser [35].

3.1. Effect of Reynolds Number

We compare the results of the WMLES with the three different boundary conditions at $Re_\tau = 5200, 10^4$ and 10^5 at grid resolution $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$. The mean streamwise velocity profile is shown in Figure 5a. Trends observed for the $Re_\tau = 5200$ case apply to higher Reynolds numbers, with D-EV providing the best results for the mean velocity profile. The overestimation of the mean velocity profile for the N-ZEV and N-EV cases, like the $Re_\tau = 5200$ case, seems to be due to the higher dissipation closer to the wall from the higher values of ν_t in the first few off-wall grid points stemming from the higher values of $\partial u/\partial y$ at the wall. The results for the turbulence intensities show similar trends as the $Re_\tau = 5200$ case for all Reynolds numbers (not shown), with D-EV showing better prediction of turbulence intensities in all three velocity components closer to the wall.

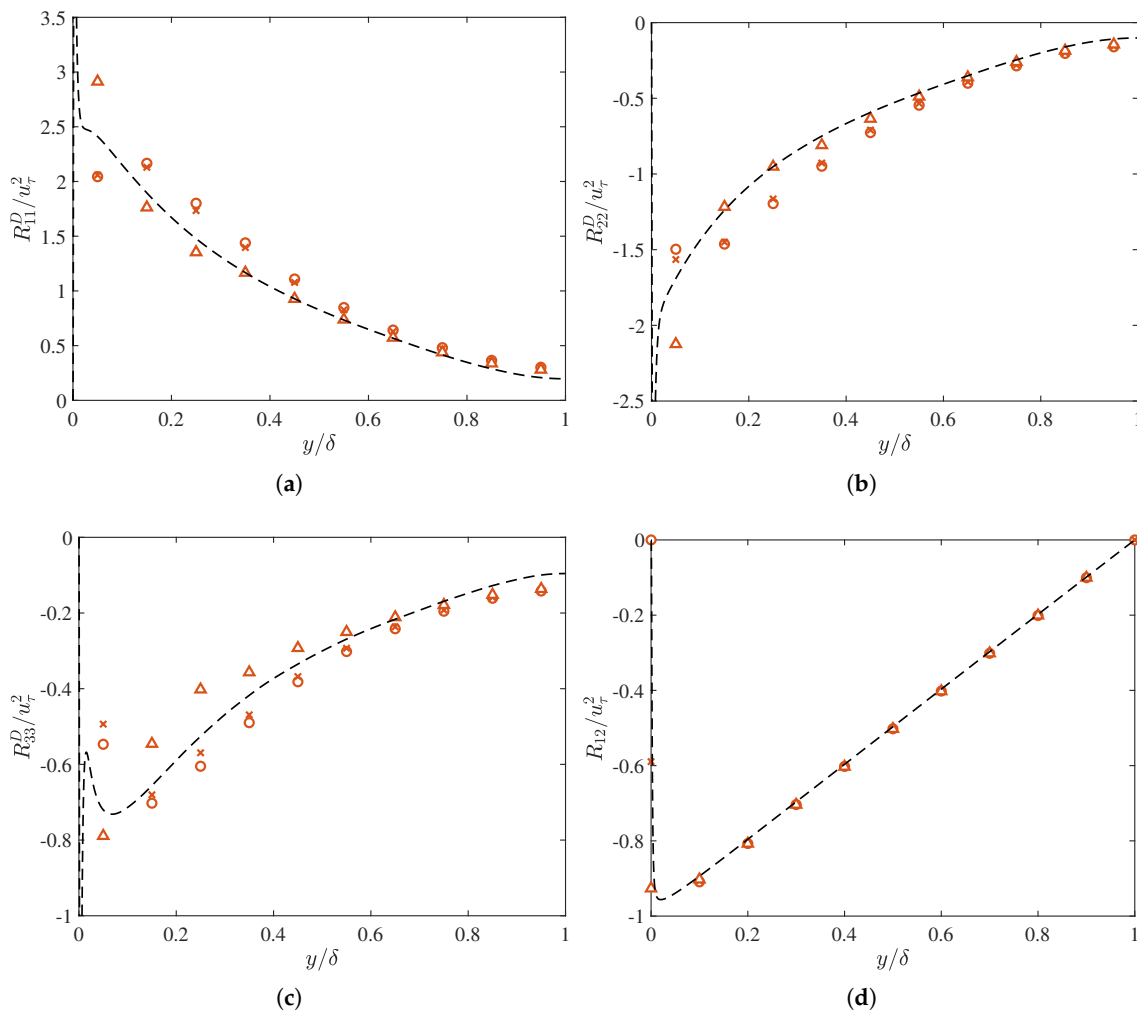


Figure 4. Deviatoric parts of the Reynolds stress tensor components: (a) R_{11}^D , (b) R_{22}^D , (c) R_{33}^D , and (d) R_{12}^D for N-ZEV (circles), N-EV (crosses), and D-EV (triangles) for the $Re_\tau = 5200$ case with $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$. The dashed line indicates the deviatoric part of the Reynolds stress tensor computed from DNS by Lee and Moser [35].

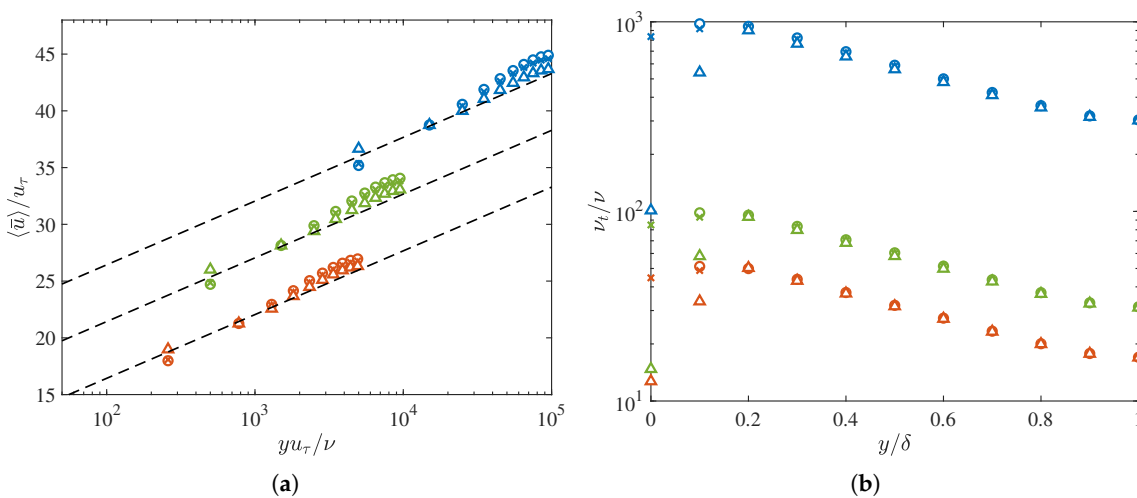


Figure 5. (a) Mean streamwise velocity profile and (b) mean eddy viscosity for N-ZEV (circles), N-EV (crosses), and D-EV (triangles) for $Re_\tau = 5200$ (red), 10^4 (green), and 10^5 (blue) with $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$. Mean velocity profiles are offset by 5 for each Reynolds number. The dashed line indicates logarithmic law $u/u_\tau = 1/\kappa \log(yu_\tau/\nu) + B$, with $\kappa = 0.41$ and $B = 5.2$.

3.2. Effect of Grid Resolution

We compare the results of the WMLES with the three different boundary conditions at $Re_\tau = 5200$ at grid resolutions $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.05, 0.1, \text{ and } 0.2$. The results obtained for the three grid resolutions are comparable for all the boundary conditions, with mean velocity profiles exhibiting similar trends for the baseline case (Figure 6a). The eddy viscosity increases for coarser grids, as the SGS contribution increases (Figure 6b). Regardless, the same observation that the eddy viscosity close to the wall is overestimated due to the large wall-normal gradient of the streamwise velocity at the wall holds for all grid resolutions. The results for the turbulence intensities show similar trends as the baseline case for all grid resolutions (not shown).

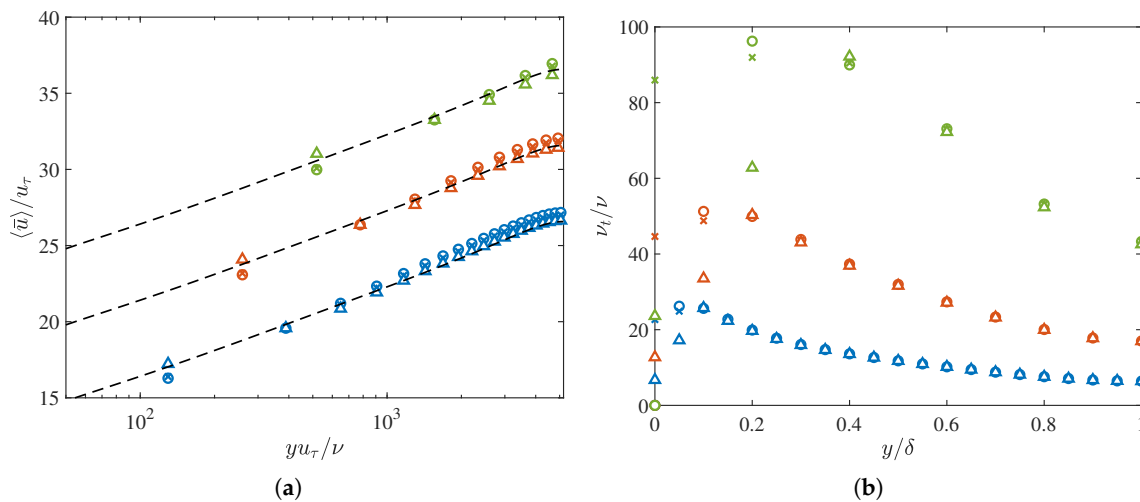


Figure 6. (a) Mean streamwise velocity profile and (b) mean eddy viscosity for N-ZEV (circles), N-EV (crosses), and D-EV (triangles) for $Re_\tau = 5200$ with $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.05$ (blue), 0.1 (red), and 0.2 (green). Mean velocity profiles are offset by 5 for each grid resolution. The dashed line indicates the DNS profile by Lee and Moser [35].

4. Application to Wall Models

To compare the effects of the two wall boundary conditions, N-ZEV and D-EV, in actual WMLES, we performed a WMLES of a channel flow at $Re_\tau = 5200$ with grid resolution $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$. The channel was driven by a constant mass flow equivalent to that of the DNS by Lee and Moser [35]. The wall shear stress was computed using the equilibrium wall model (EQWM) [14] with the first off-wall cell center as the matching location, y_m .

It is known that using N-ZEV with $y_m = \Delta_y/2$ (cell-center of the first off-wall grid point) is prone to result in the so-called log-layer mismatch [14,25]. This can be seen in Figure 7: the wall-shear stress is underestimated by 3%, resulting in a mean velocity profile that is shifted in the positive direction. On the contrary, the mean velocity profile of D-EV shows no log-layer mismatch, and the wall-shear stress is well predicted (within 0.1%). This is consistent with the observation in the previous section, showing that D-EV is able to match the mean velocity profile from the first grid-point off the wall, unlike N-ZEV.

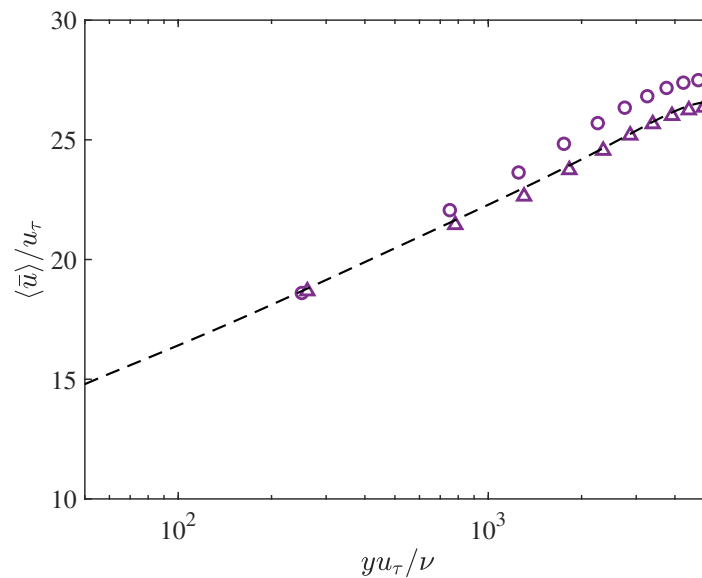


Figure 7. Mean streamwise velocity profile with EQWM ($y_m = \Delta_y/2$) with boundary conditions N-ZEV (circles) and D-EV (triangles) for $Re_\tau = 5200$ with $\Delta_x/\delta \simeq \Delta_y/\delta \simeq \Delta_z/\delta \simeq 0.1$. The dashed line indicates the DNS profile by Lee and Moser [35].

5. Conclusions

We introduced three different boundary conditions for applying wall shear stress derived from the mean stress-balance equations for a turbulent channel flow with a no-penetration boundary condition ($v|_w = 0$). Depending on whether the eddy viscosity contribution at the wall is set to zero, the traditional boundary condition used for WMLES is either the Neumann boundary condition with zero eddy viscosity at the wall (N-ZEV) or the Neumann boundary condition with nonzero eddy viscosity computed from the SGS model (N-EV). However, the mean stress-balance equation can be also satisfied by changing the eddy-viscosity be nonzero at the wall while applying a Dirichlet boundary condition for the velocities, which we call the Dirichlet boundary condition with eddy viscosity augmentation (D-EV).

The D-EV boundary condition provides improved predictions of the mean velocity profile and turbulence intensities in the first few off-wall grid points compared to N-ZEV or N-EV. This improvement can be attributed to the limited wall-normal gradient of the streamwise velocity and a smaller eddy viscosity at the wall, leading to less dissipation in this region. As most SGS models are overly dissipative in the near-wall region [36], the reduction in dissipation allows the D-EV to provide better statistics close to the wall. The three boundary conditions showed minor differences in terms of mean velocity and turbulence intensity prediction in the outer region, as expected.

The improved predictions in the first off-wall grid point for D-EV allow traditional wall models to be used with LES information from the first off-wall grid point without additional spatial or temporal filtering. This is contrary to N-EV, which is known to produce a log-layer mismatch when the first off-wall grid point is utilized as the matching location in WMLES. This was demonstrated using the equilibrium wall model [14], which showed that D-EV enables the prediction of wall shear stress within 0.1% accuracy, unlike the case with N-EV, where the wall shear stress was underpredicted by 3%. The ease of implementation of D-EV together with its more accurate predictions in WMLES using the first off-wall grid point suggest that this boundary condition might be beneficial for flow over complex geometry where information farther away from the wall may not be readily available.

Author Contributions: Conceptualization, H.J.B. and A.L.-D.; methodology, H.J.B.; software, H.J.B. and A.L.-D.; validation, H.J.B. and A.L.-D.; formal analysis, H.J.B.; investigation, H.J.B.; resources, H.J.B.; data curation, H.J.B.; writing—original draft preparation, H.J.B.; writing—review and editing, A.L.-D.; visualization, H.J.B.; supervision, H.J.B.; project administration, H.J.B.; funding acquisition, H.J.B. All authors have read and agreed to the published version of the manuscript.

Funding: This research received no external funding.

Institutional Review Board Statement: Not applicable.

Informed Consent Statement: Not applicable.

Data Availability Statement: Data available on request from the authors.

Acknowledgments: The authors would like to thank Oriol Lehmkuhl for fruitful discussions.

Conflicts of Interest: The authors declare no conflict of interest.

References

- Chapman, D.R. Computational aerodynamics development and outlook. *AIAA J.* **1979**, *17*, 1293–1313. [[CrossRef](#)]
- Choi, H.; Moin, P. Grid-point requirements for large eddy simulation: Chapman's estimates revisited. *Phys. Fluids* **2012**, *24*, 011702. [[CrossRef](#)]
- Yang, X.I.A.; Griffin, K.P. Grid-point and time-step requirements for direct numerical simulation and large-eddy simulation. *Phys. Fluids* **2021**, *33*, 015108. [[CrossRef](#)]
- Kline, S.J.; Reynolds, W.C.; Schraub, F.A.; Runstadler, P.W. The structure of turbulent boundary layers. *J. Fluid Mech.* **1967**, *30*, 741–773. [[CrossRef](#)]
- Kim, H.T.; Kline, S.J.; Reynolds, W.C. The production of turbulence near a smooth wall in a turbulent boundary layer. *J. Fluid Mech.* **1971**, *50*, 133–160. [[CrossRef](#)]
- Willmarth, W.W.; Lu, S.S. Structure of the Reynolds stress near the wall. *J. Fluid Mech.* **1972**, *55*, 65–92. [[CrossRef](#)]
- Lu, S.S.; Willmarth, W.W. Measurements of the structure of the Reynolds stress in a turbulent boundary layer. *J. Fluid Mech.* **1973**, *60*, 481–511. [[CrossRef](#)]
- Spalart, P.R.; Jou, W.H.; Strelets, M.; Allmaras, S.R.; United States Air Force Office of Scientific Research. Comments on the feasibility of LES for wings, and on a hybrid RANS/LES approach. *Adv. DNS/LES* **1997**, *1*, 4–8.
- Deardorff, J.W. A numerical study of three-dimensional turbulent channel flow at large Reynolds numbers. *J. Fluid Mech.* **1970**, *41*, 453–480. [[CrossRef](#)]
- Schumann, U. Subgrid scale model for finite difference simulations of turbulent flows in plane channels and annuli. *J. Comput. Phys.* **1975**, *18*, 376–404. [[CrossRef](#)]
- Balaras, E.; Benocci, C.; Piomelli, U. Finite-difference computations of high Reynolds number flows using the dynamic subgrid-scale model. *Theor. Comput. Fluid Dyn.* **1995**, *7*, 207–216. [[CrossRef](#)]
- Cabot, W.; Moin, P. Approximate wall boundary conditions in the large-eddy simulation of high Reynolds number flow. *Flow Turbul. Combust.* **2000**, *63*, 269–291. [[CrossRef](#)]
- Wang, M.; Moin, P. Dynamic wall modeling for large-eddy simulation of complex turbulent flows. *Phys. Fluids* **2002**, *14*, 2043–2051. [[CrossRef](#)]
- Kawai, S.; Larsson, J. Wall-modeling in large eddy simulation: Length scales, grid resolution, and accuracy. *Phys. Fluids* **2012**, *24*, 015105. [[CrossRef](#)]
- Bose, S.T.; Moin, P. A dynamic slip boundary condition for wall-modeled large-eddy simulation. *Phys. Fluids* **2014**, *26*, 015104. [[CrossRef](#)]
- Bae, H.J.; Lozano-Durán, A.; Bose, S.T.; Moin, P. Dynamic slip wall model for large-eddy simulation. *J. Fluid Mech.* **2019**, *859*, 400–432. [[CrossRef](#)]
- Piomelli, U.; Balaras, E. Wall-layer models for large-eddy simulations. *Annu. Rev. Fluid Mech.* **2002**, *34*, 349–374. [[CrossRef](#)]
- Larsson, J.; Kawai, S.; Bodart, J.; Bermejo-Moreno, I. Large eddy simulation with modeled wall-stress: Recent progress and future directions. *Mech. Eng. Rev.* **2016**, *3*, 15–00418. [[CrossRef](#)]
- Bose, S.T.; Park, G.I. Wall-modeled large-eddy simulation for complex turbulent flows. *Annu. Rev. Fluid Mech.* **2018**, *50*, 535–561. [[CrossRef](#)] [[PubMed](#)]
- Owen, H.; Radhakrishnan, S.; Gomez, S.; Lehmkuhl, O. A finite-element viscosity based non-slip wall modelling strategy for the simulation of industrial turbulent flows. *Ercraft Bull.* **2019**, *120*. (in press)
- Piomelli, U.; Ferziger, J.; Moin, P.; Kim, J. New approximate boundary conditions for large eddy simulations of wall-bounded flows. *Phys. Fluids* **1989**, *1*, 1061–1068. [[CrossRef](#)]
- Nicoud, F.; Baggett, J.S.; Moin, P.; Cabot, W. Large eddy simulation wall-modeling based on suboptimal control theory and linear stochastic estimation. *Phys. Fluids* **2001**, *13*, 2968–2984. [[CrossRef](#)]
- Lee, J.; Cho, M.; Choi, H. Large eddy simulations of turbulent channel and boundary layer flows at high Reynolds number with mean wall shear stress boundary condition. *Phys. Fluids* **2013**, *25*, 110808. [[CrossRef](#)]

24. Porté-Agel, F.; Meneveau, C.; Parlange, M.B. A scale-dependent dynamic model for large-eddy simulation: Application to a neutral atmospheric boundary layer. *J. Fluid Mech.* **2000**, *415*, 261–284. [[CrossRef](#)]
25. Yang, X.I.A.; Park, G.I.; Moin, P. Log-layer mismatch and modeling of the fluctuating wall stress in wall-modeled large-eddy simulations. *Phys. Rev. Fluids* **2017**, *2*, 104601. [[CrossRef](#)]
26. Orlandi, P. *Fluid Flow Phenomena: A Numerical Toolkit In Fluid Mechanics and Its Applications Vol. 55*; Springer: Berlin/Heidelberg, Germany, 2000.
27. Kim, J.; Moin, P. Application of a fractional-step method to incompressible Navier–Stokes equations. *J. Comput. Phys.* **1985**, *59*, 308–323. [[CrossRef](#)]
28. Wray, A.A. Minimal-Storage Time Advancement Schemes for Spectral Methods; Technical Report; NASA Ames Research Center: Mountain View, CA, USA, 1990.
29. Rozema, W.; Bae, H.J.; Moin, P.; Verstappen, R. Minimum-dissipation models for large-eddy simulation. *Phys. Fluids* **2015**, *27*, 085107. [[CrossRef](#)]
30. Bae, H.J.; Lozano-Durán, A.; Bose, S.T.; Moin, P. Turbulence intensities in large-eddy simulation of wall-bounded flows. *Phys. Rev. Fluids* **2018**, *3*, 014610. [[CrossRef](#)]
31. Lozano-Durán, A.; Bae, H.J. Characteristic scales of Townsend’s wall-attached eddies. *J. Fluid Mech.* **2019**, *868*, 698–725. [[CrossRef](#)] [[PubMed](#)]
32. Lozano-Durán, A.; Bae, H.J. Error scaling of large-eddy simulation in the outer region of wall-bounded turbulence. *J. Comput. Phys.* **2019**, *392*, 532–555. [[CrossRef](#)]
33. Park, G.I.; Moin, P. An improved dynamic non-equilibrium wall-model for large eddy simulation. *Phys. Fluids* **2014**, *26*, 37–48. [[CrossRef](#)]
34. Lozano-Durán, A.; Jiménez, J. Effect of the computational domain on direct simulations of turbulent channels up to $Re_\tau = 4200$. *Phys. Fluids* **2014**, *26*, 011702. [[CrossRef](#)]
35. Lee, M.; Moser, R.D. Direct numerical simulation of turbulent channel flow up to $Re_\tau \approx 5200$. *J. Fluid Mech.* **2015**, *774*, 395. [[CrossRef](#)]
36. Jimenez, J.; Moser, R.D. Large-eddy simulations: Where are we and what can we expect? *AIAA J.* **2000**, *38*, 605–612. [[CrossRef](#)]